

Implement the following problem in a MATLAB *script* m-file.

SIMPLE SECOND-ORDER ELECTRICAL SYSTEM

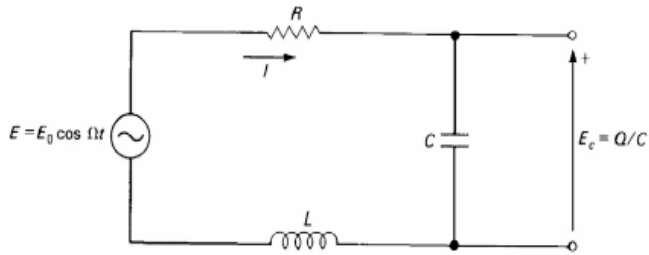
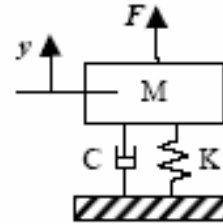


Figure RLC



Mechanical Second Order System

Figure 1: Left: Series RCL circuit. Right: Mass-spring-damper system. In this problem, at time $t = 0$, voltage V is applied (i.e., a switch is closed), or force F is suddenly applied. These input signals are held constant “for all time.”

The following discussion is based on **Figure 1**: a series RCL circuit, and a mass-spring-damper system. The two systems are mathematically equivalent to each other. We will assume that the system is **underdamped**... which means when subjected to a constant input, it will oscillate about its final configuration for some time before reaching equilibrium.

The system is initially at “rest” (current is zero; velocity is zero) and there is no energy stored (no current through the inductor and no voltage across the capacitor; no extension in the spring). A step input is suddenly applied at time $t = 0$ (a switch is closed to apply DC voltage V_s ; a constant force F is suddenly applied).

The response of an **underdamped** system to a step input is:

$$x(t) = u_0 \left[1 - e^{-\zeta \omega_n t} (\cos \omega_d t - \gamma \sin \omega_d t) \right]$$

Response:	$x(t)$ (capacitor voltage v_c , displacement $y(t)$)
Size of step:	u_0 (applied voltage V_s , or static displacement, F/k)
Undamped Natural Frequency:	$\omega_n = \sqrt{\frac{1}{LC}}$ (electrical), $\omega_n = \sqrt{\frac{k}{m}}$ (mechanical)
Damping Ratio:	$\zeta = \frac{R}{2} \sqrt{\frac{C}{L}}$ (electrical), $\zeta = \frac{c}{2\sqrt{km}}$ (mechanical)
Damped Natural Frequency:	$\omega_d = \omega_n \sqrt{1 - \zeta^2}$
Normalizing Constant:	$\gamma = \frac{\zeta}{1 - \zeta^2}$

The equation for $x(t)$ is only valid for **underdamped** system. For an underdamped system: $\boxed{\zeta < 1}$

The function $x(t)$ is a constant (the response’s final value), plus an exponentially decaying sinusoid. The system oscillates about its final value; the oscillation eventually dies down.

For **electrical systems with the components in series (Figure 1)**, u_o is the applied DC voltage V_s , while $x(t)$ is the voltage measured across the capacitor, $v_c(t)$, so:

$$v_c(t) = V_s \left[1 - e^{-\zeta \omega_n t} (\cos \omega_d t - \gamma \sin \omega_d t) \right]$$

The final value of v_c is V_s . In your electrical engineering textbooks, you may have used the **neper frequency**, or **damping factor** $\alpha = \zeta \omega_n$ in the exponential term. A circuit is underdamped when $\alpha < \omega_n$ or $\zeta < 1$.

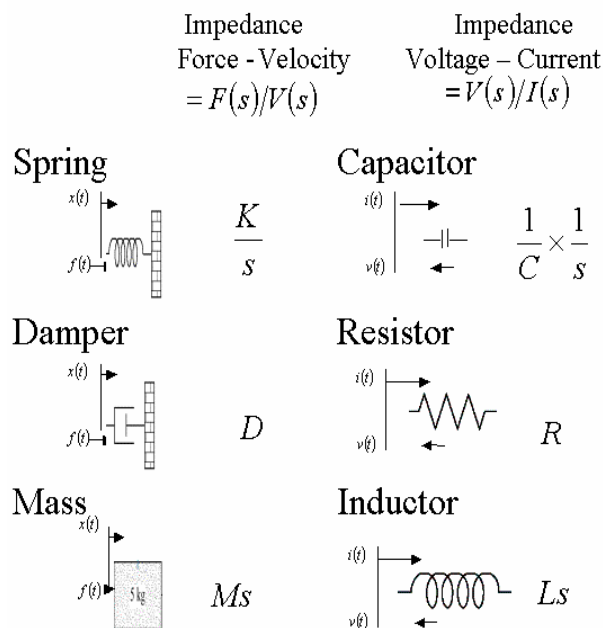
For **mechanical systems (Figure 1)**, u_o is the static deflection of the spring, F/k , the applied force divided by the spring constant. $x(t)$ is the spring displacement, $y(t)$, so:

$$y(t) = \frac{F}{k} \left[1 - e^{-\zeta \omega_n t} (\cos \omega_d t - \gamma \sin \omega_d t) \right]$$

The final value of y is F/k .

Systems that are **overdamped** ($\zeta > 1$ or $\alpha > \omega_n$) or **critically damped** ($\zeta = 1$ or $\alpha = \omega_n$) have different responses, beyond the scope of this exercise.... Perhaps it is for a future assignment.

Component Equivalence



Source Nise 2004

Figure 2. Component Equivalence.

EXERCISE:

Evaluate the system response for:

$$\begin{aligned}\omega_n &= 104 \text{ rad/sec} & \zeta &= 0.3 \\ u_0 &= 1 \text{ unit (Volts or Force/stiffness)} & 0 \leq t \leq 0.4 \text{ (sec)}\end{aligned}$$

Sample code is given below.

Physical question:

What improvements could be made to the system response? How would you change the parameters?

Programming Question:

Can you develop a code where you interactively input R , C and L (or c , k and m)?

Can you use `if-elseif-end` statements to create a program that can analyze **underdamped**, **overdamped** and **critically damped** systems for any (practical) combinations of R , C and L (or D , K and M).

```
%Engr 179 Allan Hancock College
%Sherman Wiggin
%Spring 2005
%Response of an underdamped second-order system to a step input

omegan=104 %The natural frequency
zeta = 0.3 %The dampening ratio
u0=1 %Input step size = unity
omegad=omegan*sqrt(1-zeta^2)%The natural damped frequency
gamm_a=zeta/sqrt(1-zeta^2)%non-dimensional parameter (simplifies
notation)
t=linspace(0,0.4) %Creates 100 data points 0 to 0.4
x=u0*(1-exp(-zeta*omegan*t).*(cos(omegad*t)-
gamm_a*sin(omegad*t)))
plot(t,x)
```